



Northern Illinois University

DESIGN AND MECHANICAL STABILITY ANALYSIS OF THE INTERACTION REGION FOR THE INVERSE COMPTON SCATTERING GAMMA-RAY SOURCE USING FINITE ELEMENT METHOD

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Thesis defense

7/5/2017

Contents

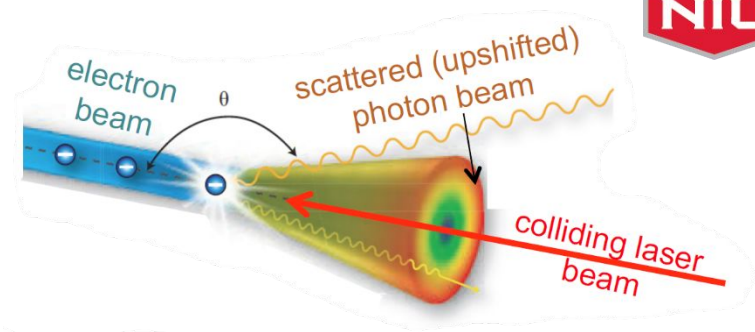


- Introduction
- Design
- Static analysis
- Modal analysis
- Harmonic analysis
- Conclusion

Introduction - ICS



Inverse Compton Scattering – process of upshifting low frequency photons by colliding them with relativistic electron bunches. ICS is most effective in the head-on collision, when θ is close to 180° . Resulting radiation has a donut shape and $1/\gamma$ angle of propagation.



$$E_\gamma \approx 4\gamma^2 E_L$$

$$E_\gamma = h\nu$$

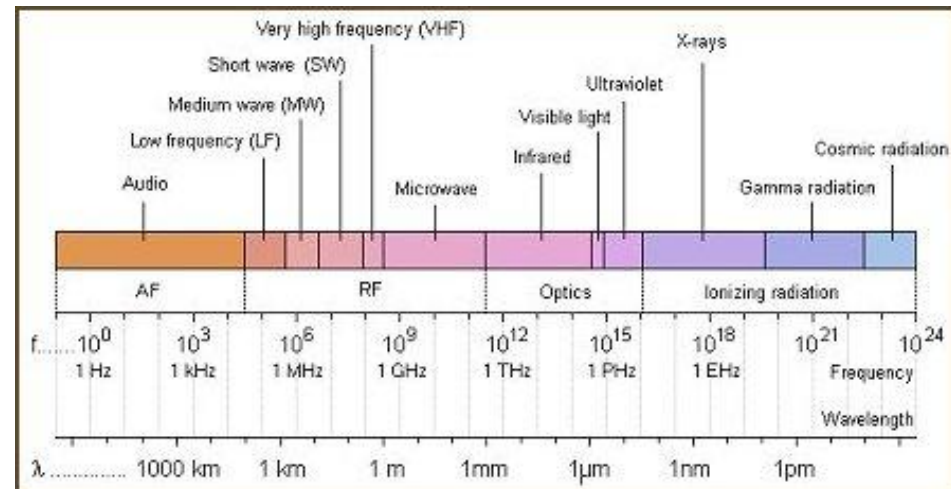
γ - Lorentz factor

h - Plank constant

E_γ - Energy of the upshifted photon

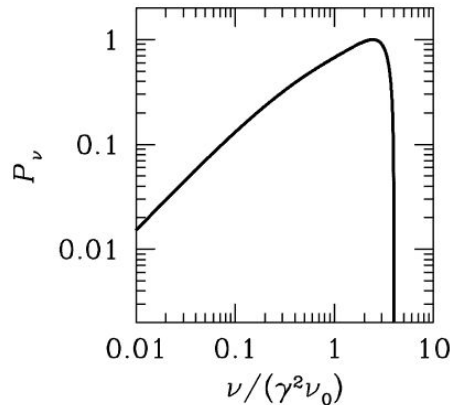
E_L - Initial energy of the photon

ν - Frequency of the upshifted photon

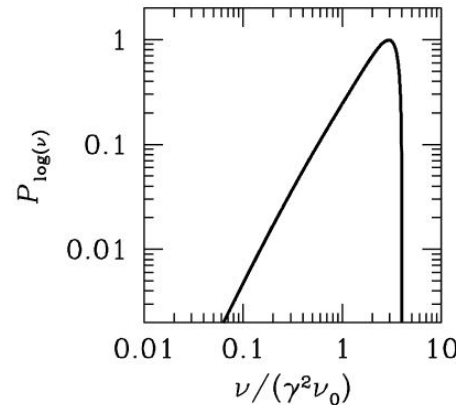


$$1 \text{ MeV} = 2.42 \times 10^{20} \text{ Hz}$$

Introduction - ICS



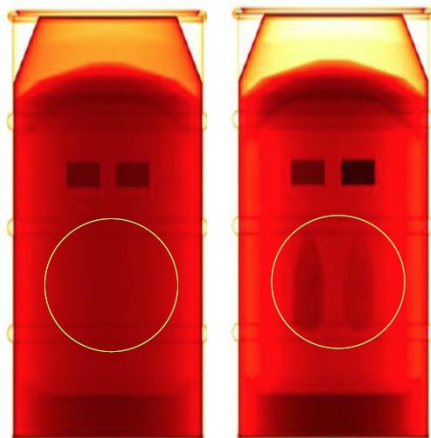
Monte-Carlos simulation of γ -ray imaging
using two different γ -ray sources



The Inverse Compton spectrum of electrons with energy γ irradiated by photons of frequency ν_0 . The log-log plot of power per logarithmic frequency range (right) more accurately shows how peaked the spectrum is. This explains why X and γ radiation generated by ICS has a relatively high *Brilliance*.

$$\text{brilliance} = \frac{\text{photons}}{\text{second} \cdot \text{mrad}^2 \cdot \text{mm}^2 \cdot 0.1\% \text{ BW}}$$

Gamma rays produced by ICS are monoenergetic with small relative bandwidth (below 1 %) and offer high photon flux. Finally, they do not include the interaction with any solid target and therefore are in principle scalable to high repetition rate as no heat management is involved.



2 MeV
Bremsstrahlung
gamma rays

1.7 MeV
Laser Compton
gamma rays

Image from C. Barty, LLNL, 2008

Introduction - Applications

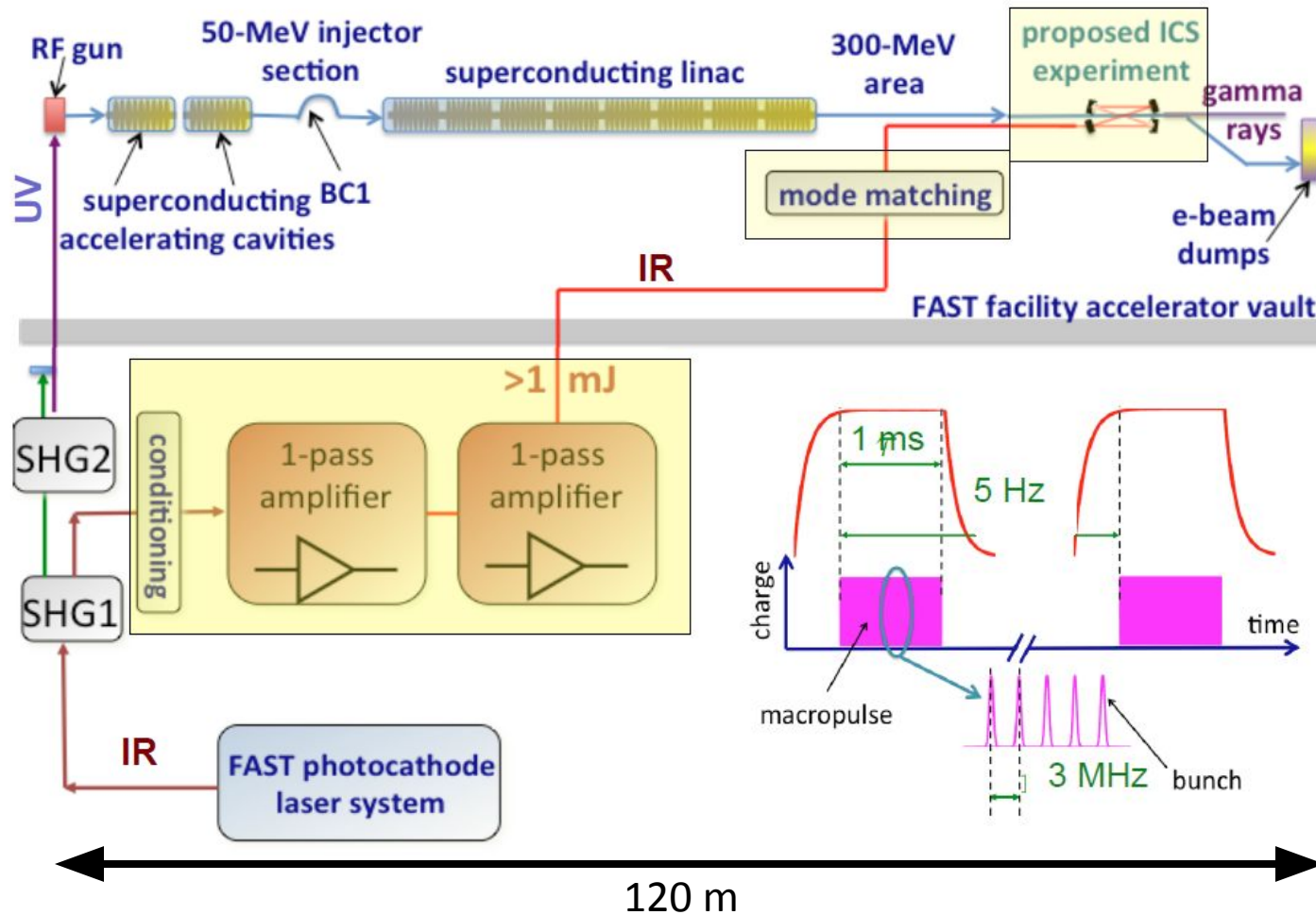


- Standoff inspection
- Nuclear element detection
- Oncology
- Nuclear astrophysics
- Nuclear medicine



Domestic Nuclear
Detection Office

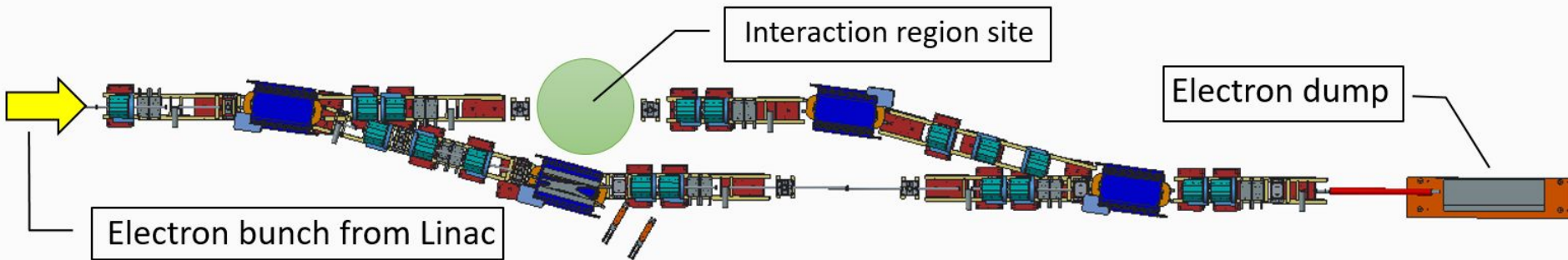
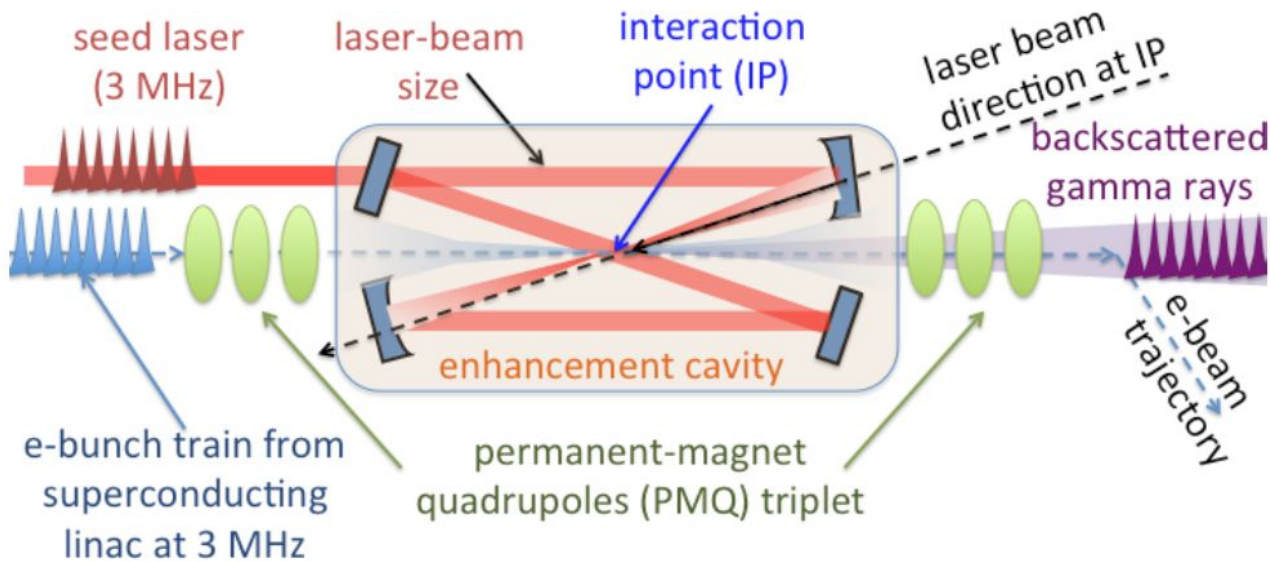
Introduction - FAST



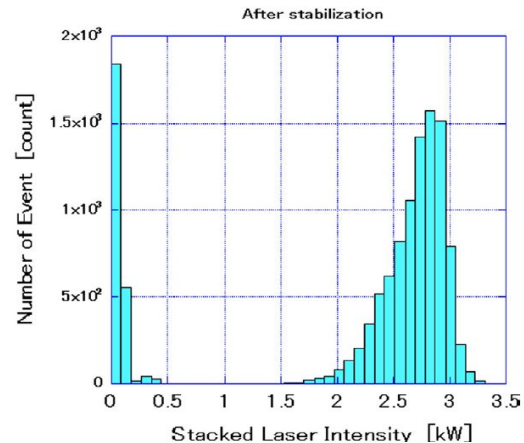
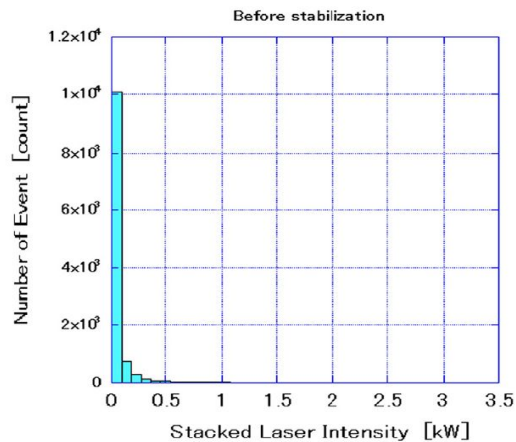
Introduction - Interaction region



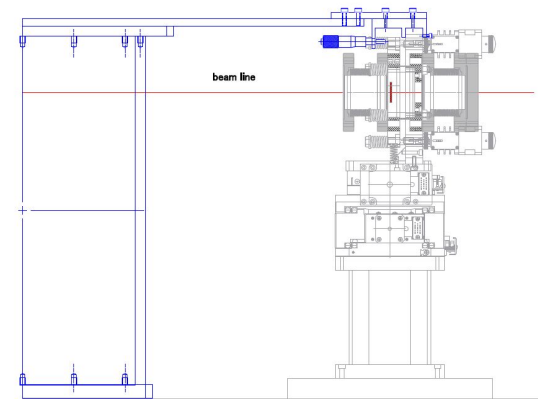
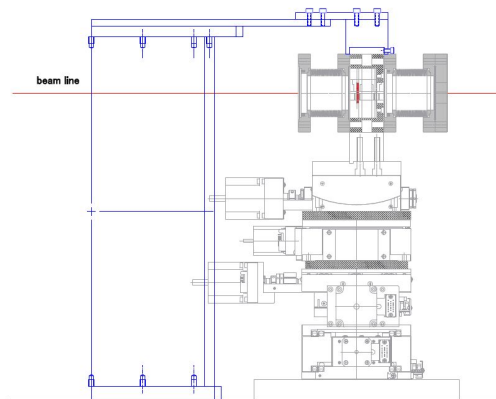
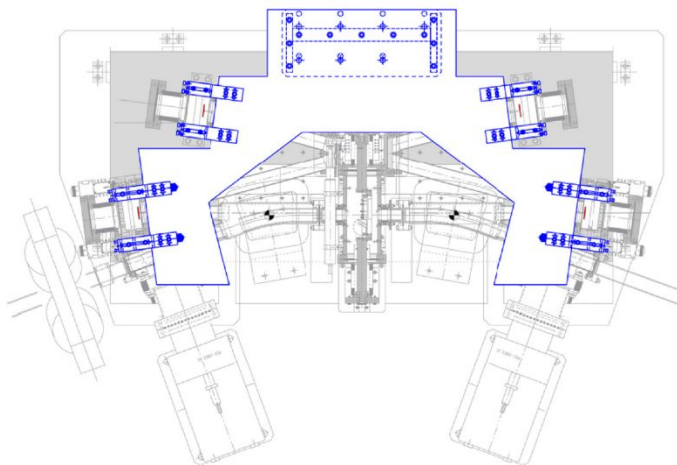
Concept of the interaction region



Introduction - Main challenge



Histograms of the stacked laser intensity. Left – prior to the improvement of the stability, right – after the improvement



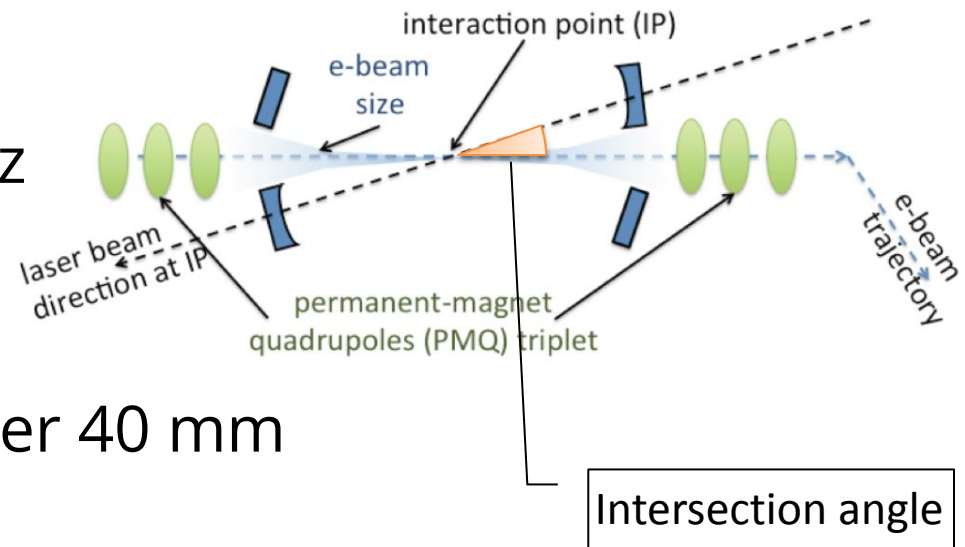
Hiroataka Shimizu - "Development of a 4-mirror optical cavity for an inverse Compton scattering experiment in the STF" KEK, 1-1 Oho, Tsukuba, Ibaraki 305-0801, Japan

Design - Objective



Cavity requirements:

- Recirculation cavity
- Target finesse > 1000
- Vacuum chamber
- Impulse frequency 3 MHz
- No bending magnets
- Intersection angle $\phi < 5^\circ$
- Focusing magnet diameter 40 mm
- Setup length < 1.5 m
- Electron line height over the floor 1200 mm



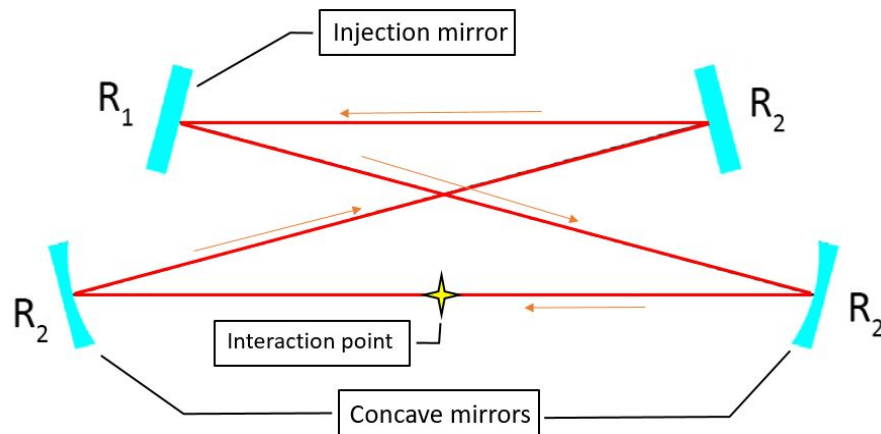
Design - Finesse



Finesse is a characteristic of oscillatory systems and resonators.

$R_1 = 99.9\%$ (entrance mirror)

$R_2 = 99.995\%$ (high reflectivity mirror)



$$F = \frac{\pi^4 \sqrt{R_1 R_2^3}}{1 - \sqrt{R_1 R_2^3}}$$

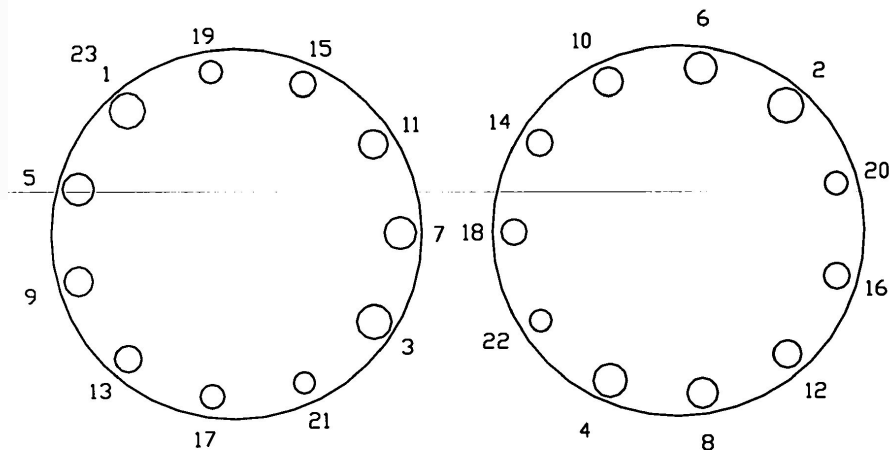
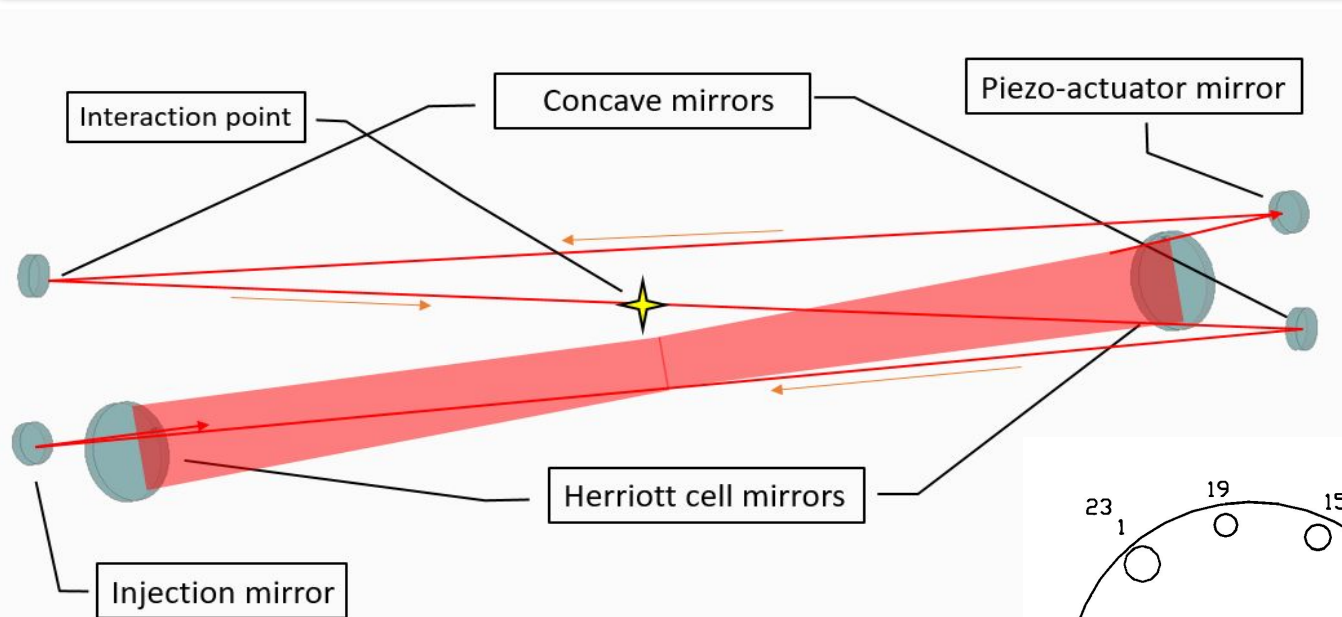
$F \sim 5500$ at v matching
the optical path length

$$F = \frac{\pi^4 \sqrt{(R_1 R_2^3)^k}}{1 - \sqrt{(R_1 R_2^3)^k}}$$

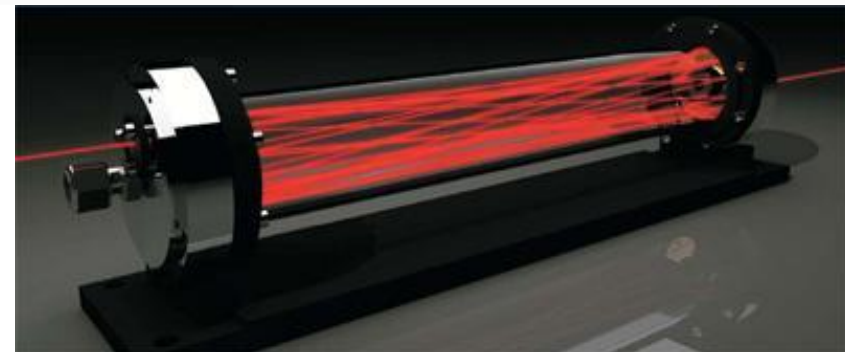
$F \sim 200$ at $k=27$
(number of round trips)

Planar bow-tie optical setup (H. Shimizu)

Design - Herriott cell



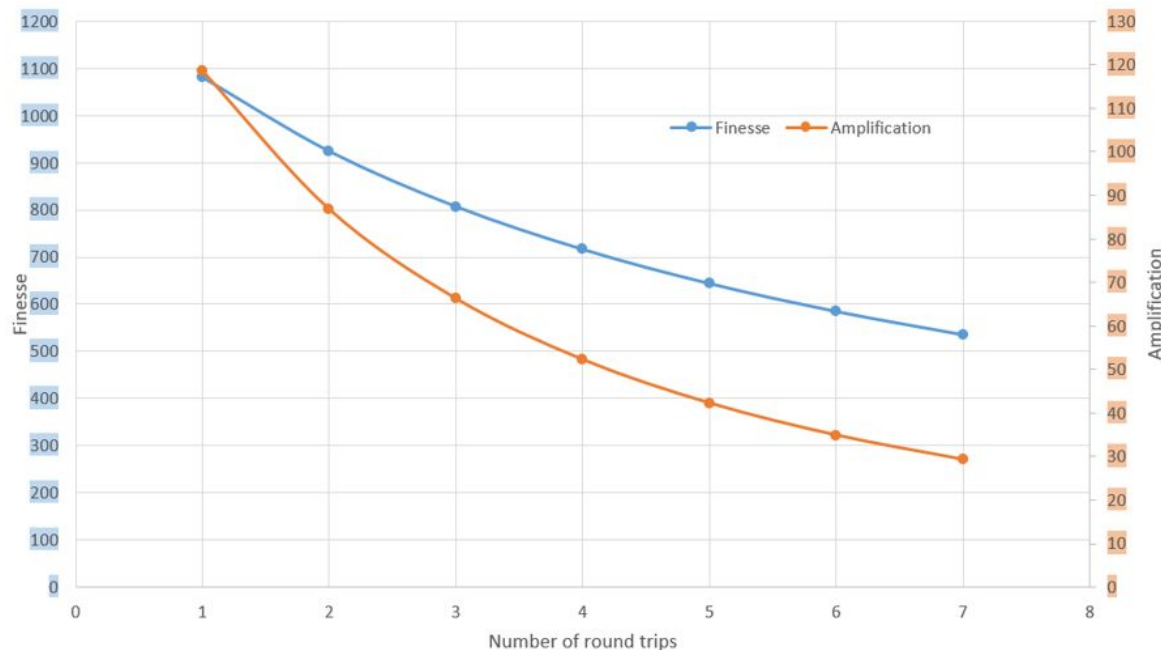
Francesco D'amato - "Variable length Herriott-type multipass cell", EP 1972922 A1



Design - Finesse and amplification estimates



Optical path length of one trip	100	50.0	33.3	25.0	20.0	16.7	14.3
k (number of round trips)	1	2	3	4	5	6	7
n (total reflection number)	93.3	45.0	28.9	20.8	16.0	12.8	10.5
a (optical amplitude loss)	0.99710	0.99661	0.99611	0.99562	0.99513	0.99464	0.99415
Finesse	1080	924	807	716	644	585	536
Finesse decrease %	0%	14%	25%	34%	40%	46%	50%
Amplification	119	87	66	52	42	35	29
Amplification decrease %	0%	27%	44%	56%	64%	71%	75%

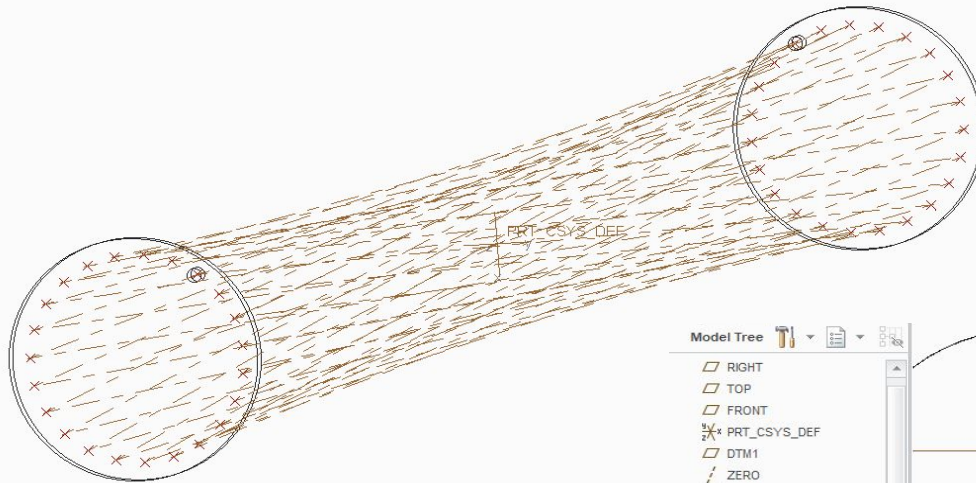


$$a = \sqrt{R_1^k R_2^{k(n+3)}}$$

$$F = \frac{\pi\sqrt{a}}{1-a}$$

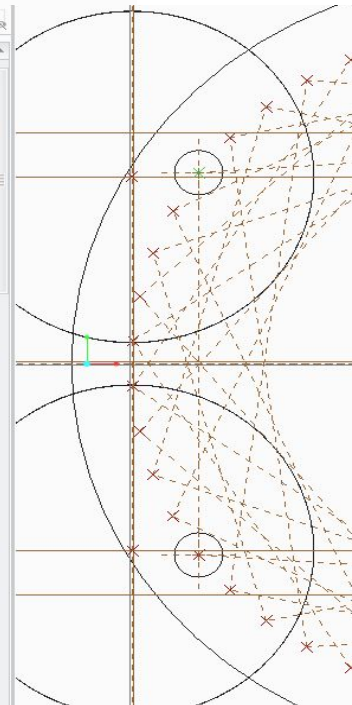
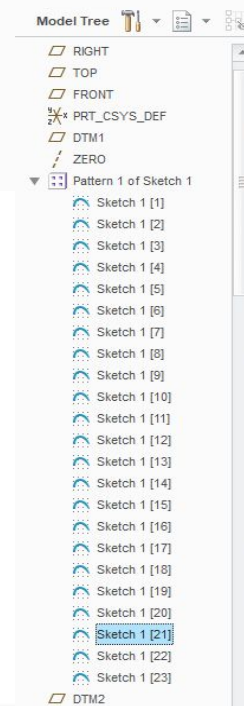
$$A = \frac{1-R_1}{(1-a)^2}$$

Design - Herriott cell



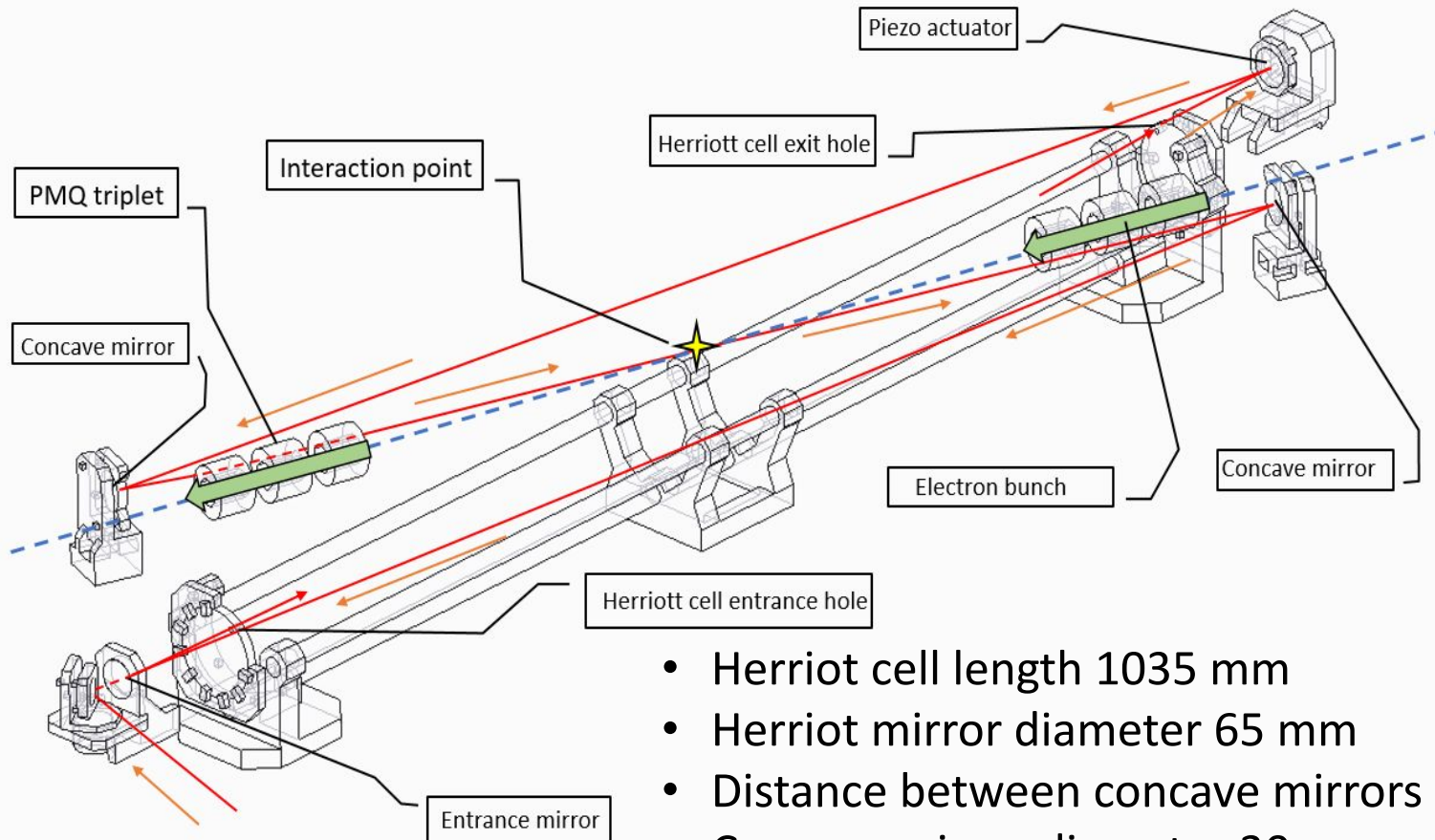
$$\alpha = 360^\circ/23 = 15.65^\circ$$

$$f_H = \frac{L}{2(1-\cos\alpha)} = 13.4 \text{ m}$$



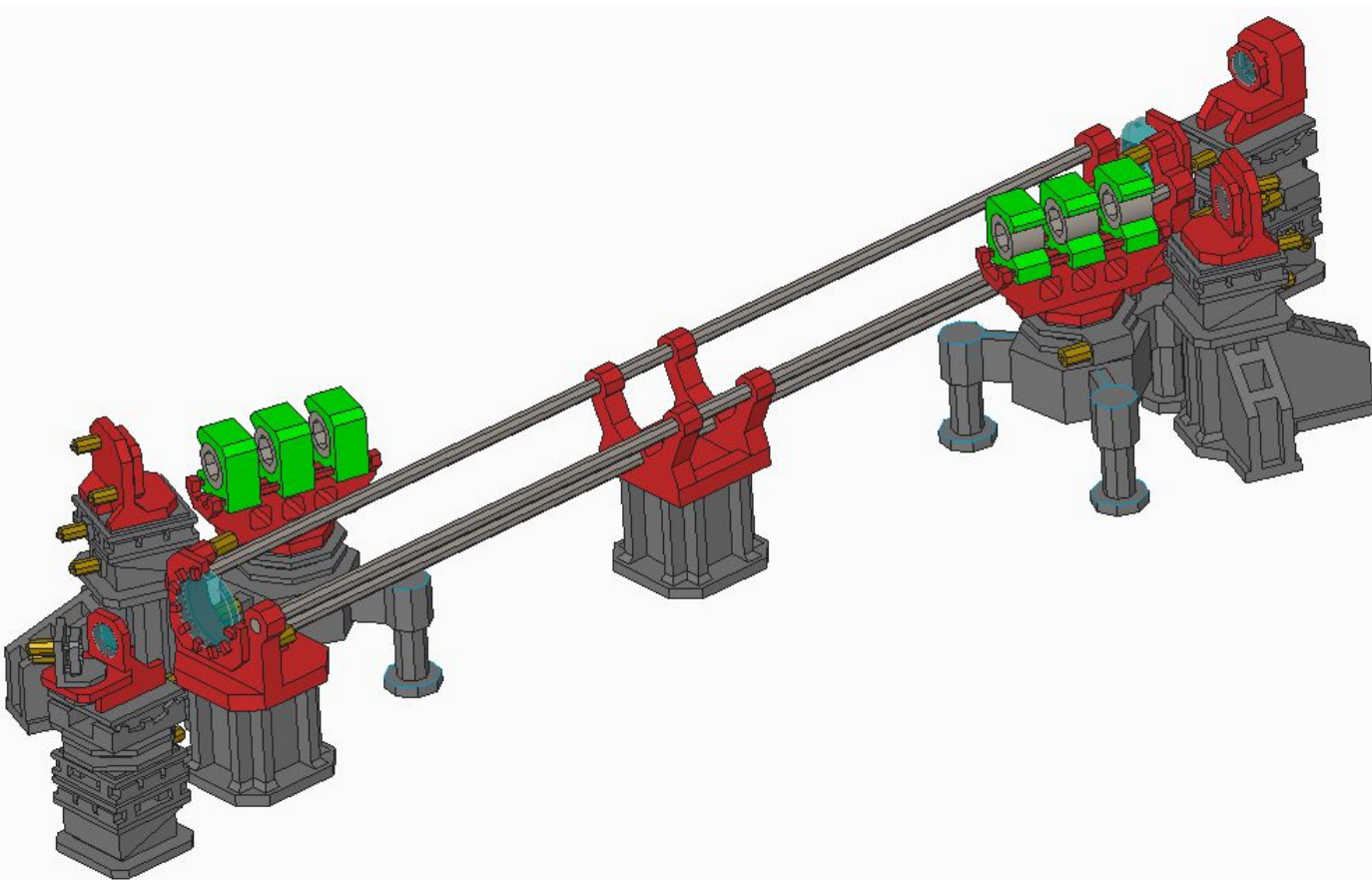
	Length	Multiplier
Herriott cell (mm)	1035.05	45
Herriot-reflector (mm)	122.396	2
Reflector-concave (mm)	1104.48	2
Concave-concave (mm)	969	1
Herriott path (m)	46.57725	
Outside path (mm)	3423	
Total path (mm)	50000.002	
Difference (mm)	0.00	

Designing - Dimensions



- Herriot cell length 1035 mm
- Herriot mirror diameter 65 mm
- Distance between concave mirrors 969 mm
- Concave mirror diameter 30 mm
- Electron and laser beam intersection angle 5°

Design - mounts and supports

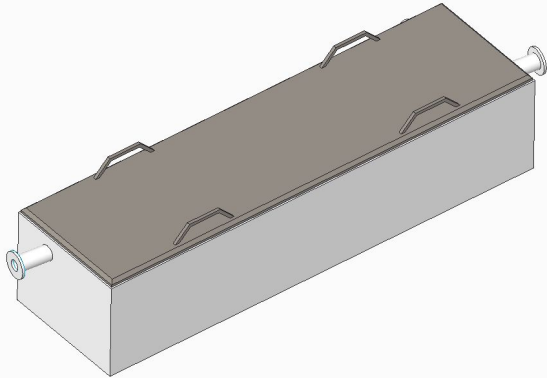


Number of individual models - 33

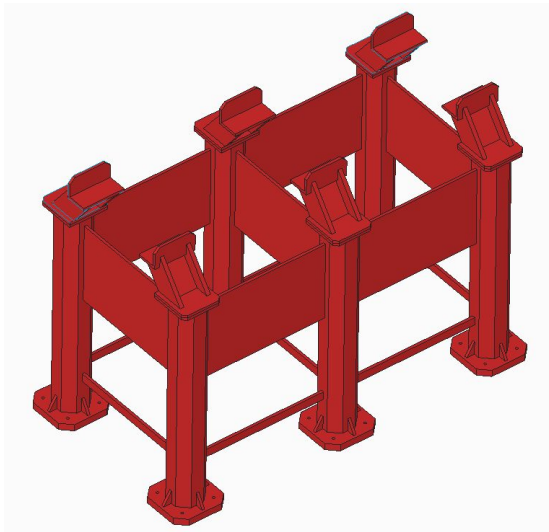
Number of assembly elements - 108

Build version - 3.12

Design - Vacuum chamber and frame

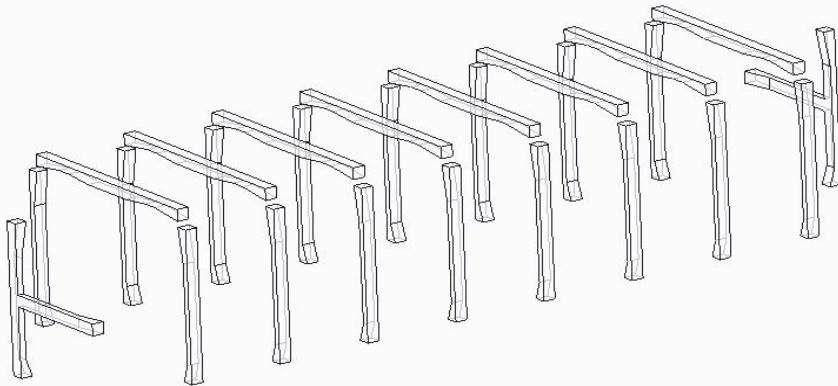
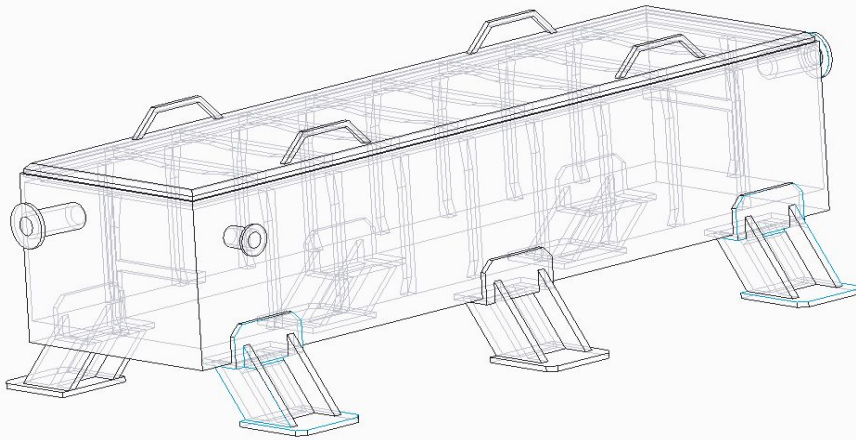


Dimensions: 1500x420x336 mm
Weight: 280 kg



Dimensions: 1400x1015x780 mm
Weight: 321 kg

Static analysis - Implosion test



The von Mises yield criterion

The von Mises stress is often used in determining whether an isotropic and ductile metal will yield when subjected to a complex loading condition. This is accomplished by calculating the von Mises stress and comparing it to the material's yield stress, which constitutes the von Mises Yield Criterion.

Static analysis - Implosion test



ANSYS stress units - MPa

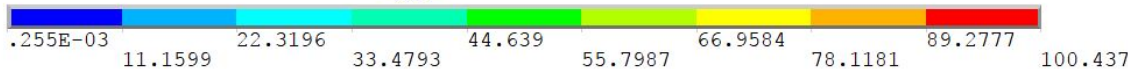
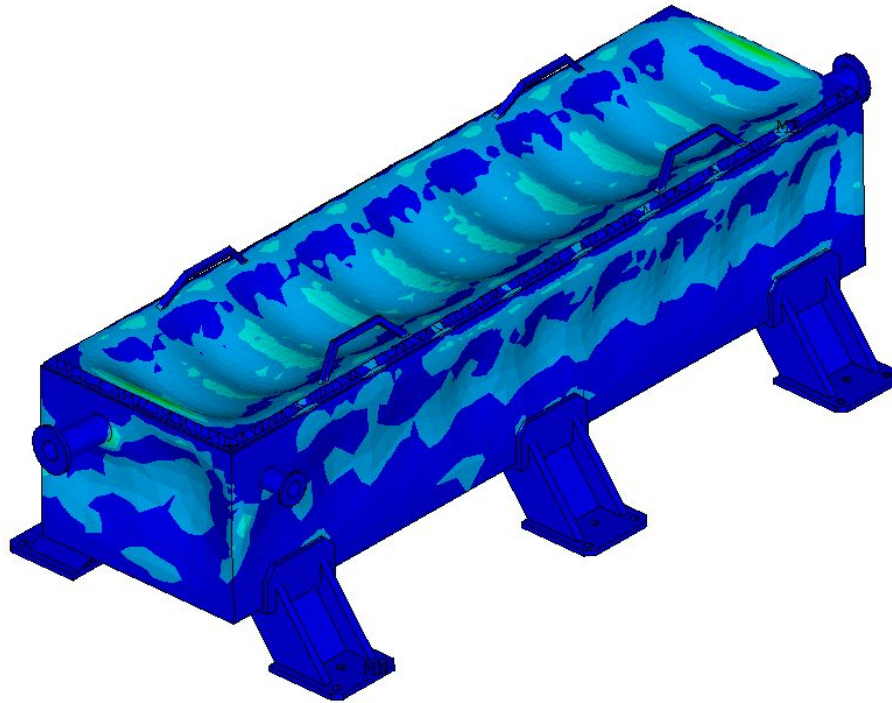
A36 steel properties:

Density of 7,800 kg/m³

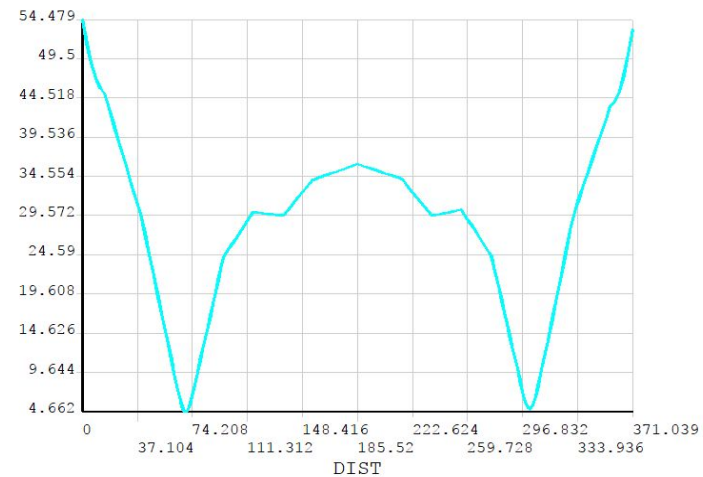
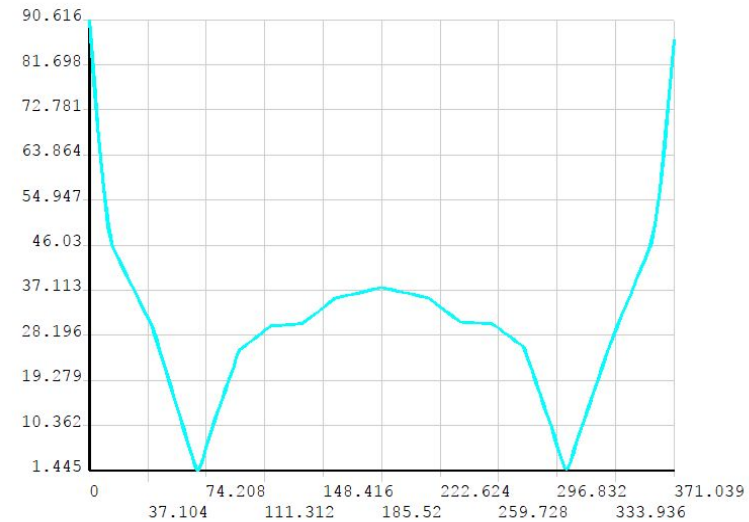
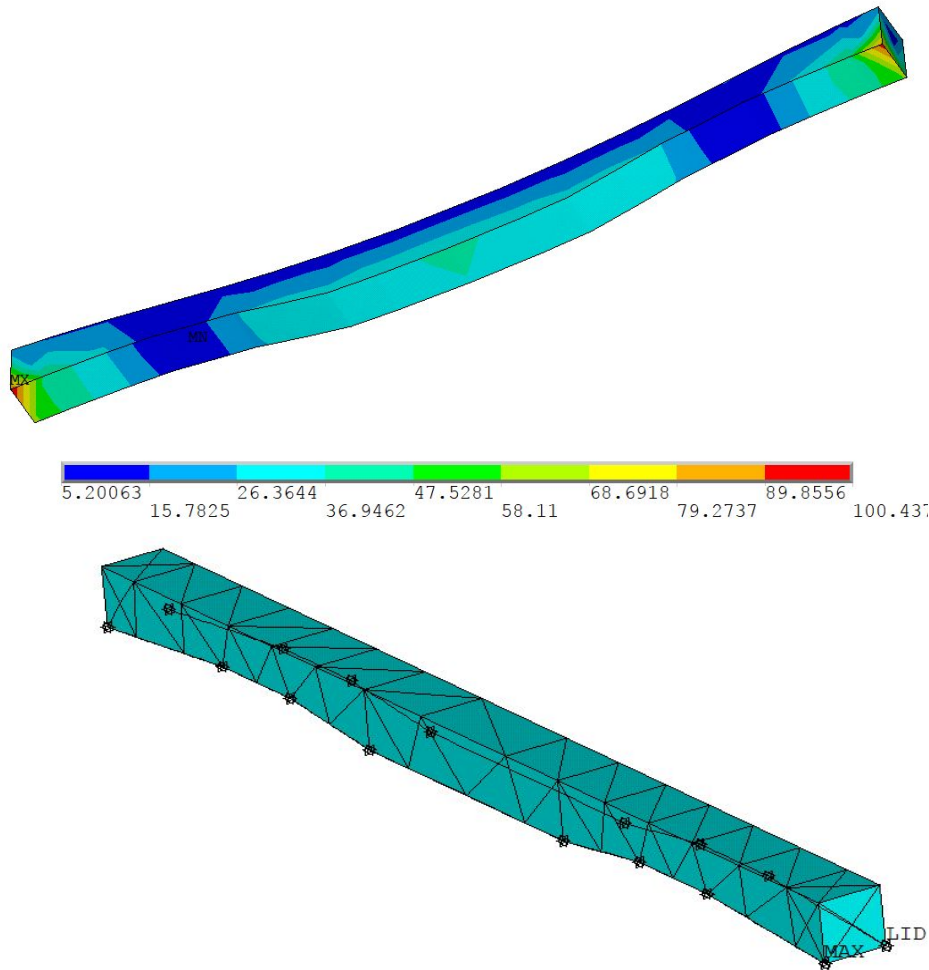
Young's modulus 200 GPa

Poisson's ratio of 0.26

A36 steel in plates, bars, and shapes with a thickness of less than 8 in (203 mm) has a minimum yield strength of 36,000 psi (250 MPa)



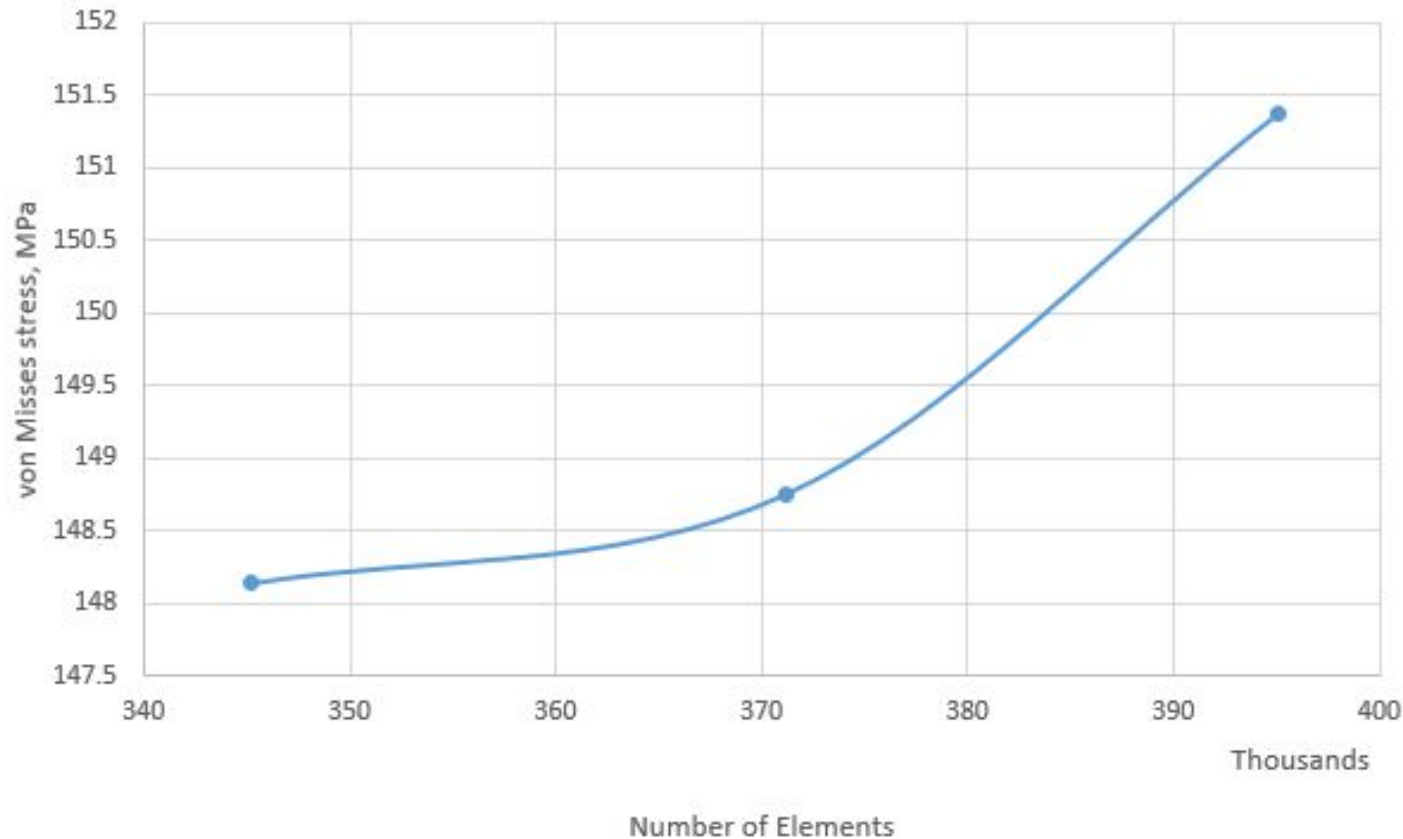
Static analysis - Implosion test



Static analysis - Convergence

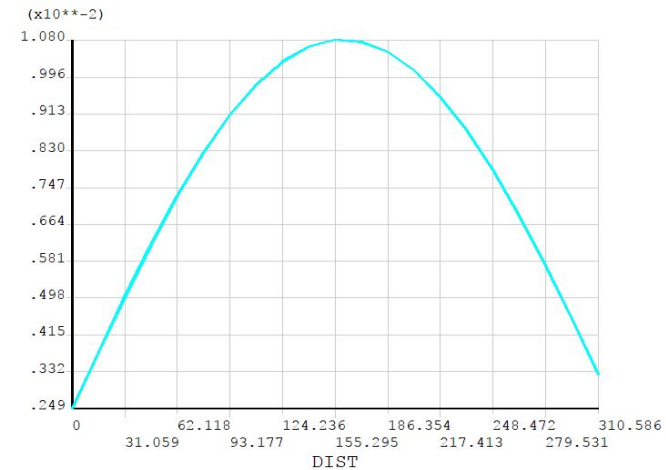
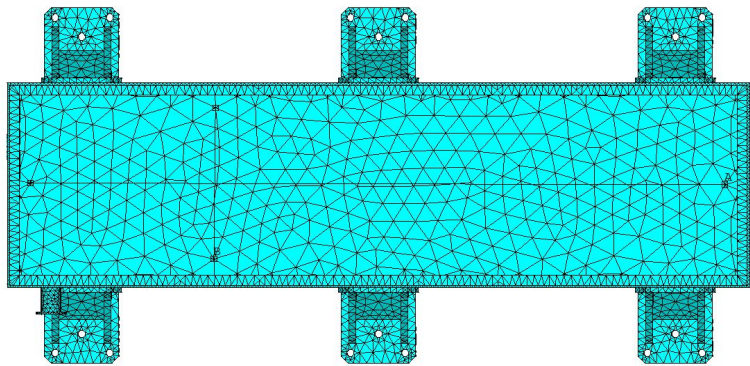
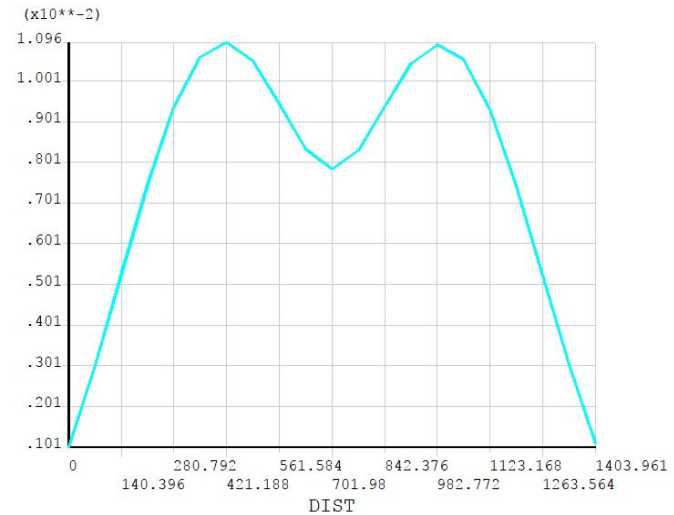
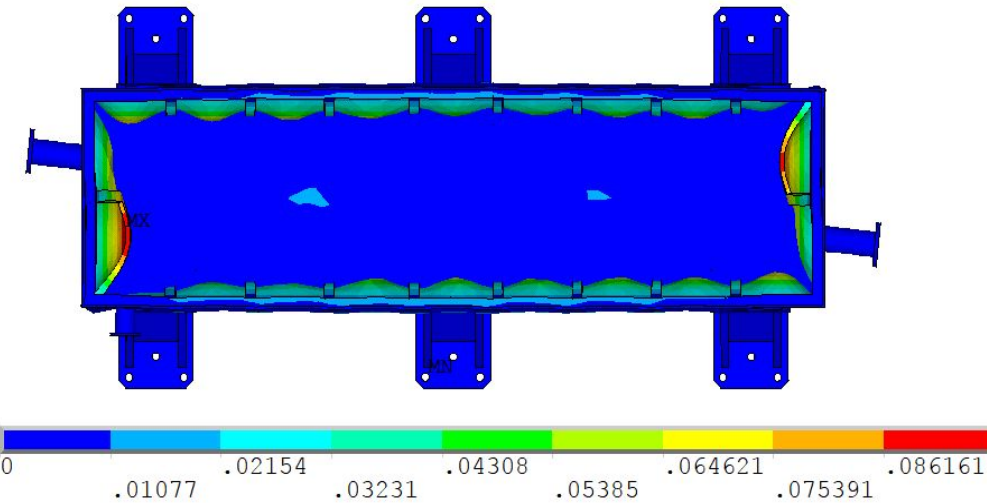


von Mises stress convergence



Von Mises stress at singularity points does not converge and grows with higher mesh resolutions

Static analysis - Displacement



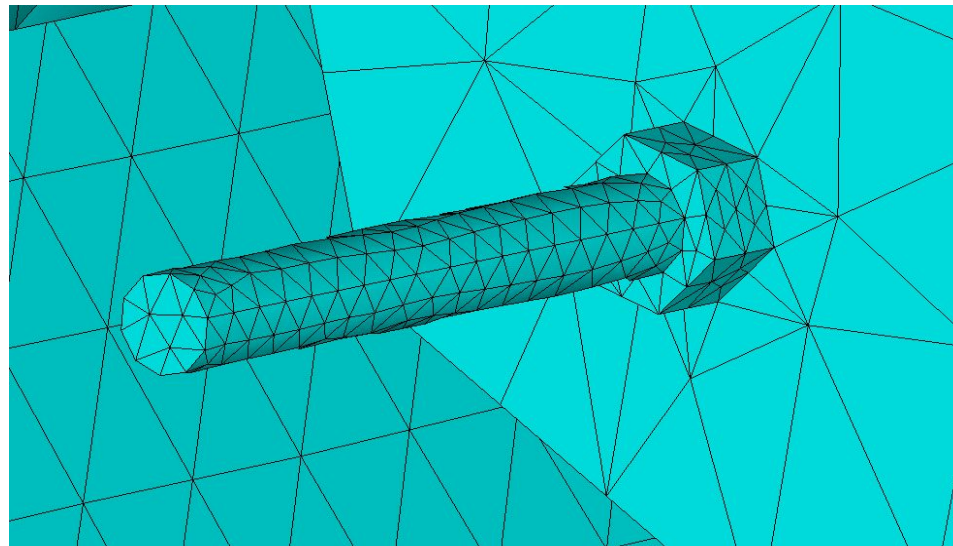
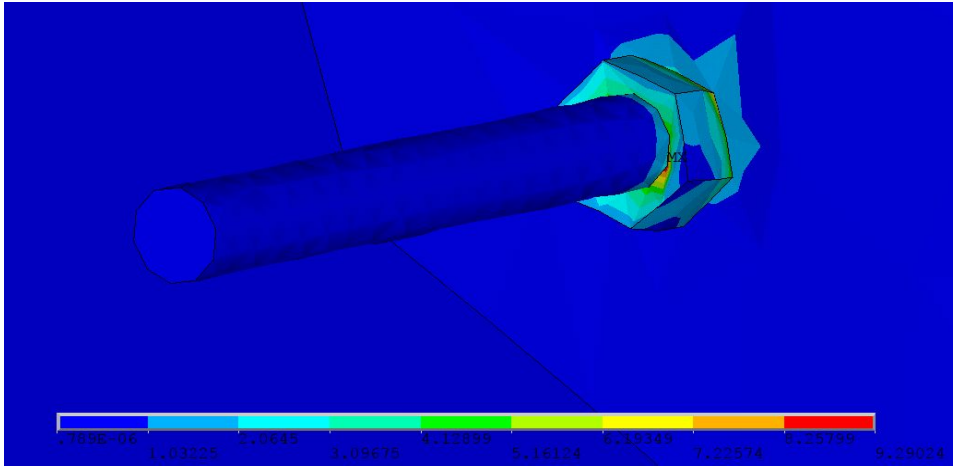
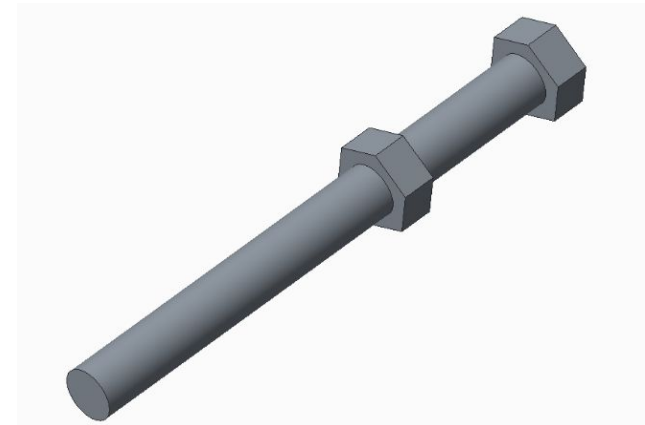
Static analysis - Gravity compression



Von Mises stress - 9.29 MPa

Generally, the stands are fastened hard to the floor with 3/8" bolts into drop-in inserts. Main frame is mounted to the floor by 24 hexagonal bolts (4 per each of six legs)

	Body Diameter Basic	Width Across Flats Basic	Head Height Basic
inches	3/8	9/16	1/4
mm	9.53	14.28	6.35



Modal analysis



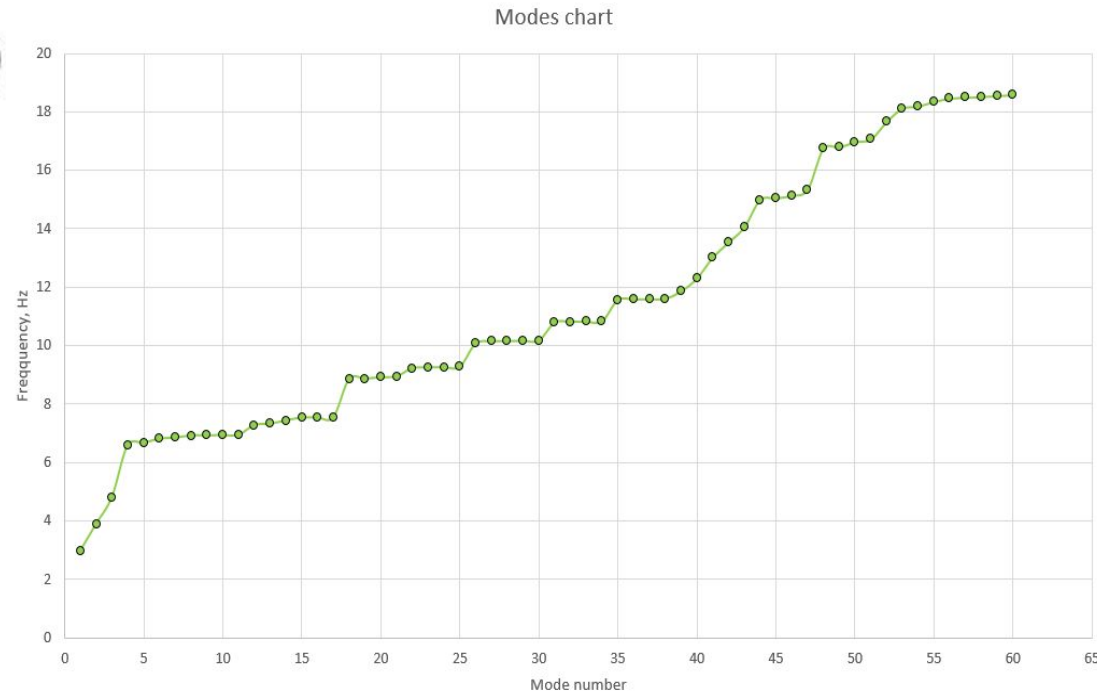
The purpose of performing a modal analysis is to find the natural frequencies and mode shapes of a structure. If a structure is going to be subjected to vibrations, then it is important to analyze where the natural frequencies occur so that the structure can be designed appropriately

$$[M]\{\ddot{u}\} + [K]\{u\} = 0$$

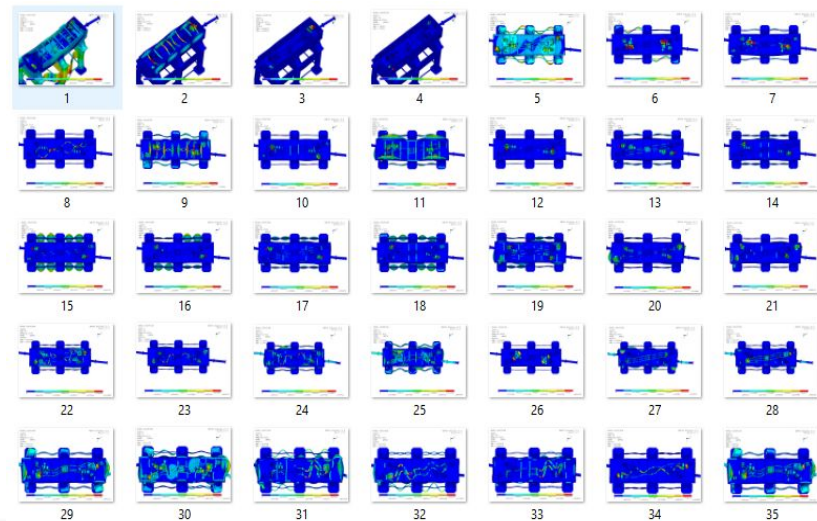
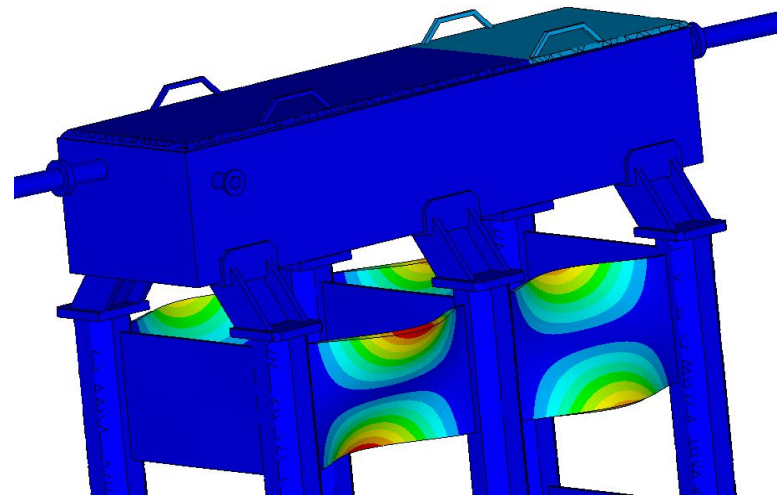
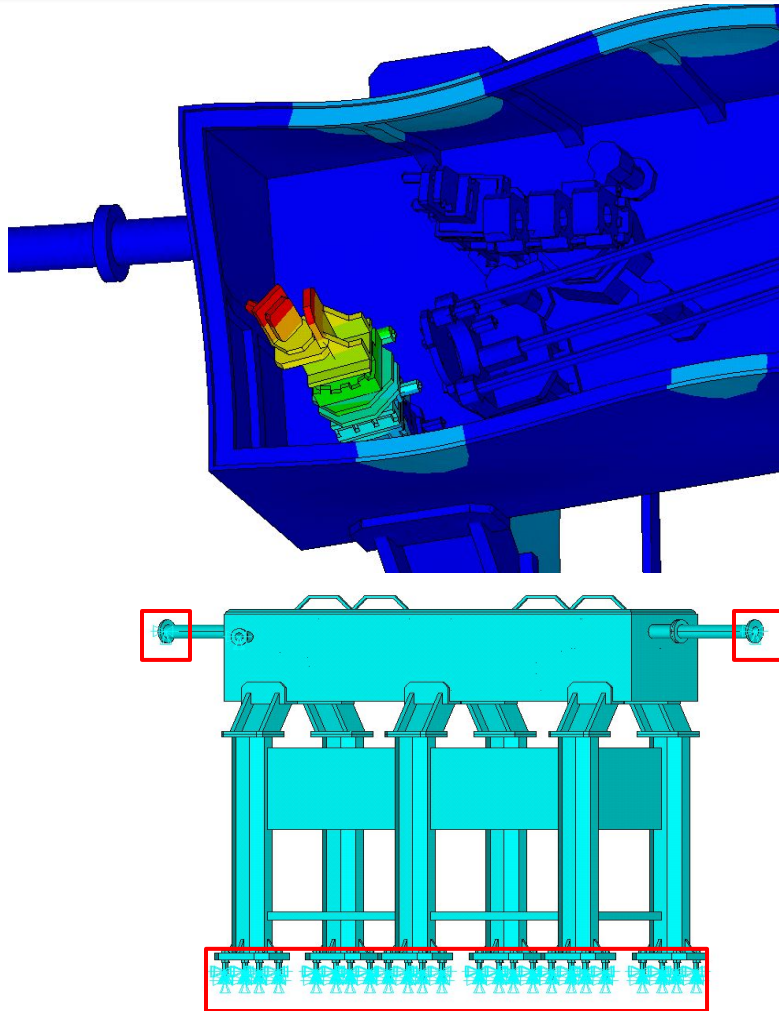
$$\{u\} = \{\varphi\}_i \cos(\omega_i t)$$

$$|[K] - \omega^2[M]| = 0$$

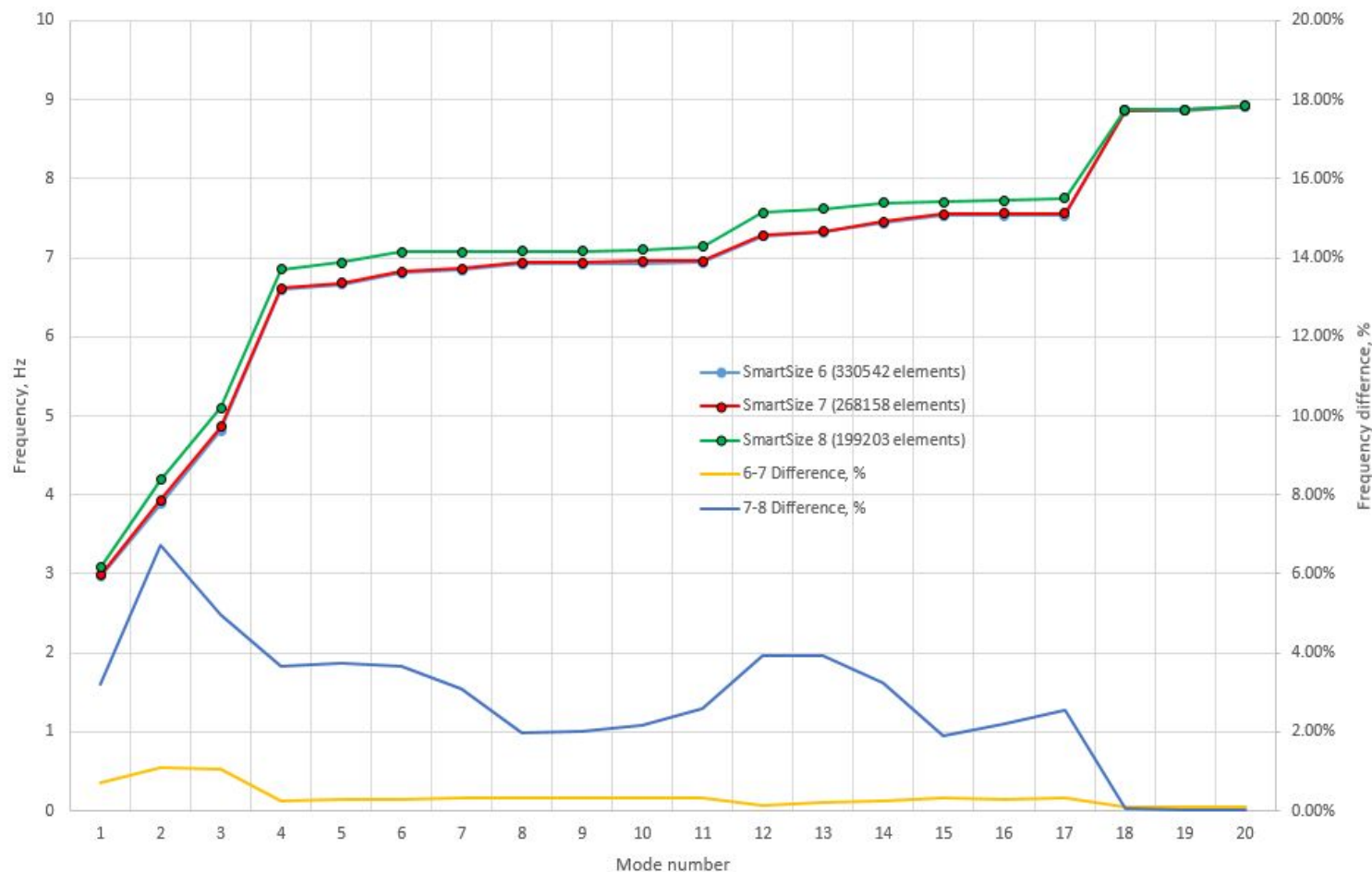
$$f_i = \frac{\omega_i}{2\pi}$$



Modal analysis - Modal maps



Modal analysis - Convergence



Harmonic analysis - Full



A harmonic analysis finds the steady state response of a structure under sinusoidal loading conditions. A harmonic, or frequency-response, analysis considers loading at one frequency only. Loads may be out-of-phase with one another, but the excitation is at a known frequency. This procedure is not used for an arbitrary transient load.

$$[M]\{\ddot{u}\} + [C]\{\dot{u}\} + [K]\{u\} = \{F^a\}$$

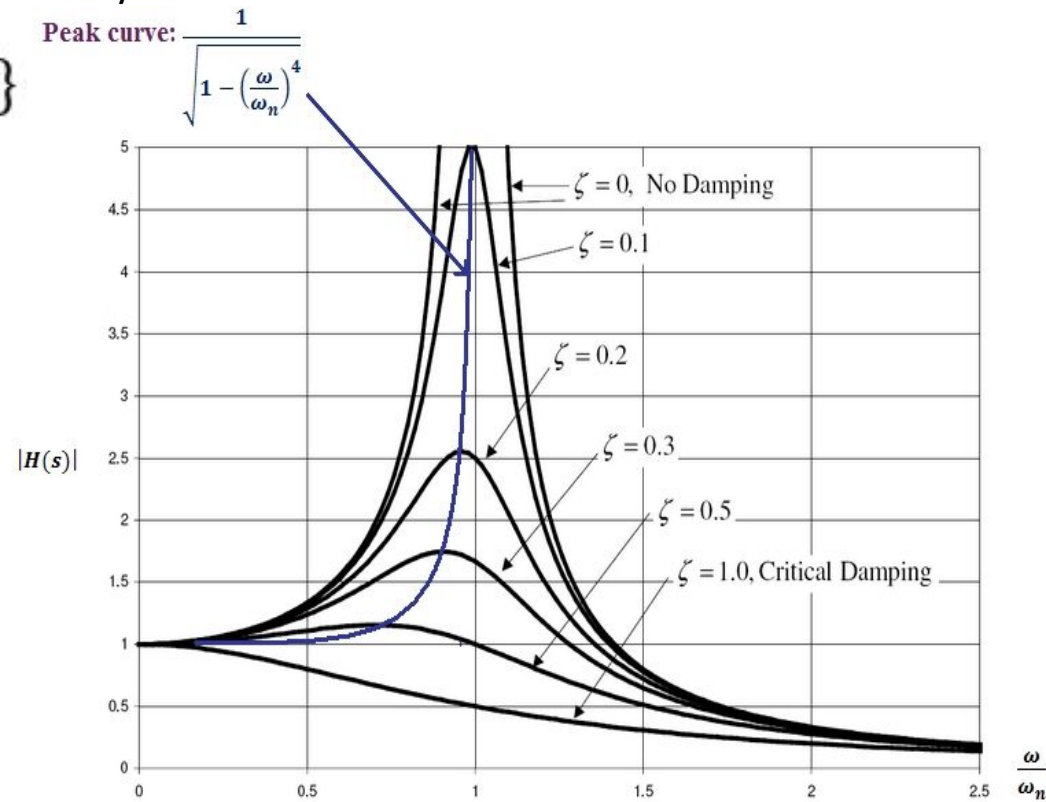
$$[C] = \alpha [M] + \beta [K]$$

$$\xi_j = \frac{\alpha}{2\omega_j} + \frac{\beta\omega_j}{2}$$

$$\xi_j = \frac{c}{c_{cr}}$$

Types of damping available in Full harmonic analysis:

- Alpha damping
- Beta damping
- Constant damping ratio



Harmonic analysis - Loading data



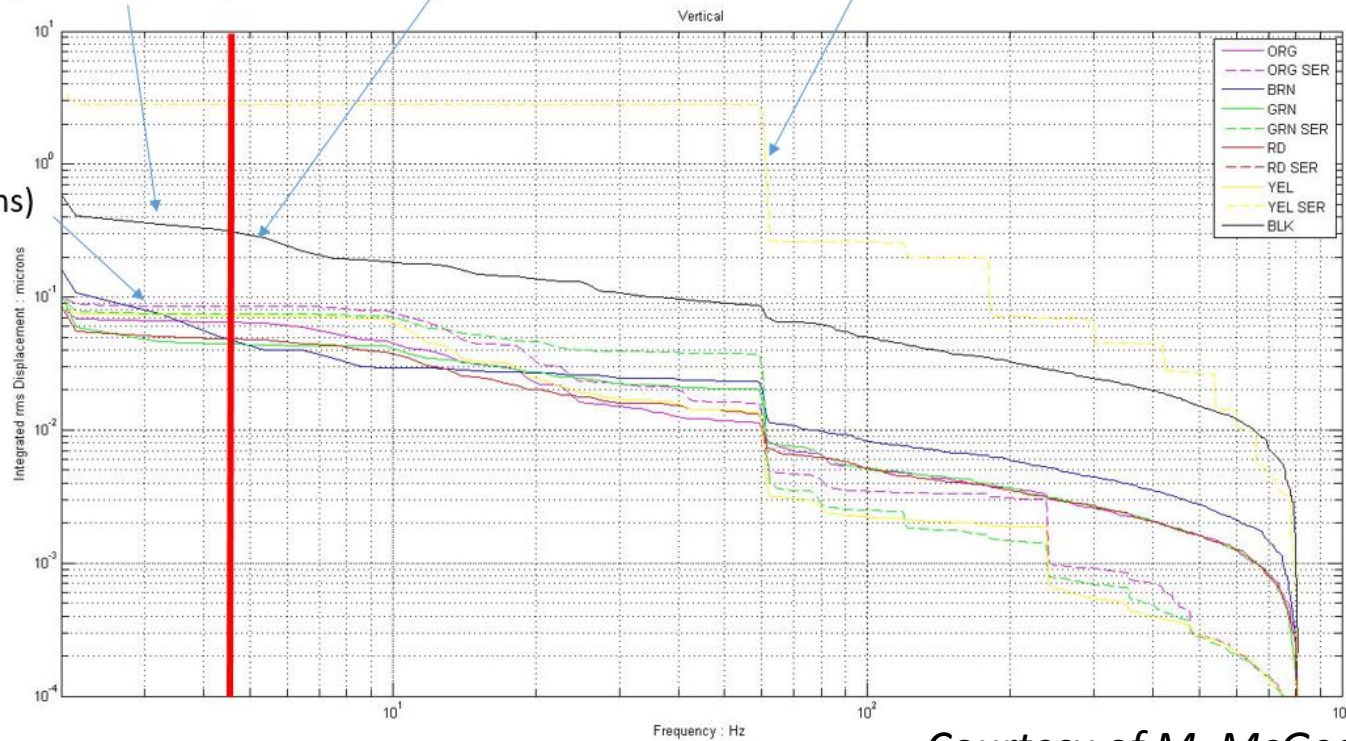
Vertical Integrated Displacement (rms) Results 8 June 2017 (12:00 – 13:00) FAST

Elevated (on 80/20 rails) $\sim 0.3 \mu\text{m}$ (rms)

Geophone roll-off (below 4.5 Hz)

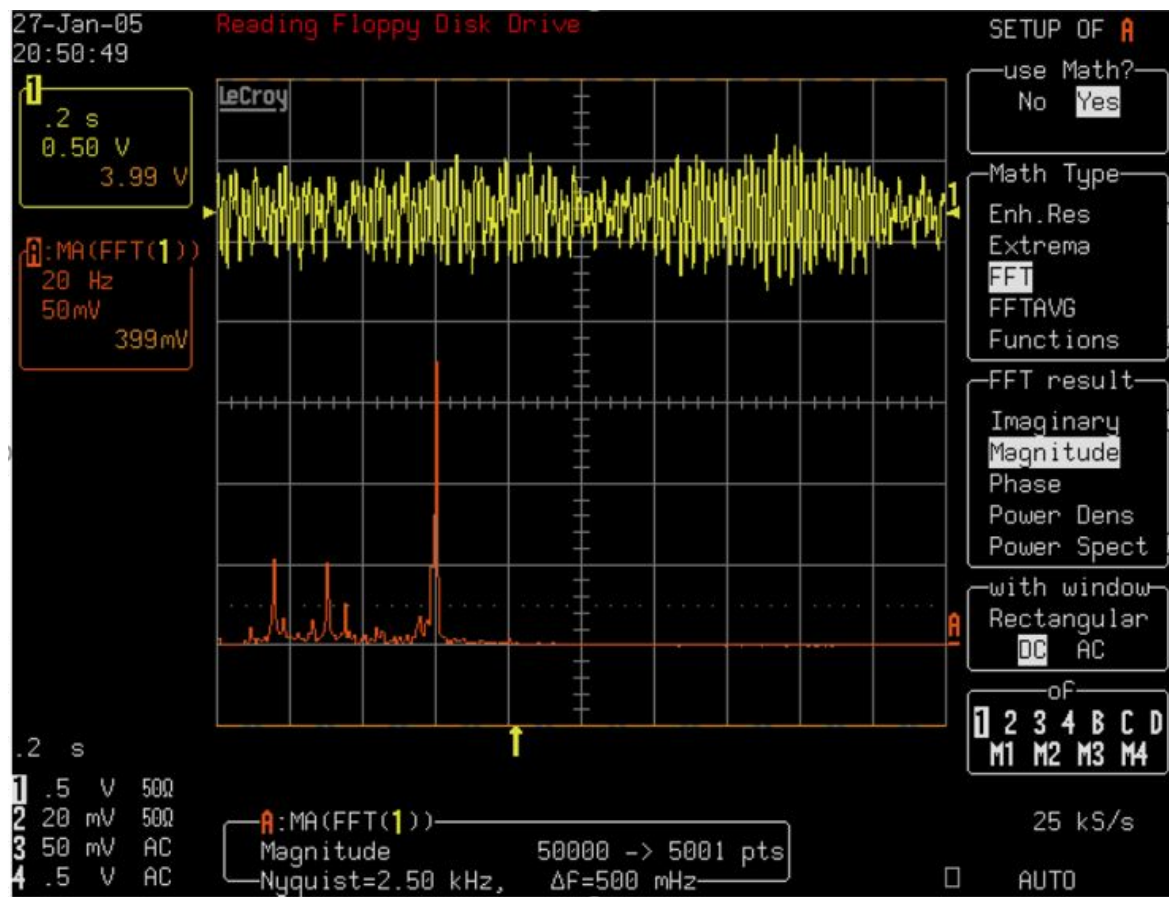
60 Hz (electrical)

Avg. $\sim 0.1 \mu\text{m}$ (rms)



Courtesy of M. McGee (Fermilab)

Harmonic analysis - Seismograph readings



Fourier transform is used to convert signal from time domain to frequency domain. Calculating a Fourier transform requires understanding of integration and imaginary numbers.

$$F(\omega) = \int_{-\infty}^{\infty} f(t)e^{-j\omega t} dt$$

$|F(\omega)|$ is called the amplitude spectrum of f

$$F(\omega) = \int_{-\infty}^{\infty} f(t) \cos \omega t dt - j \int_{-\infty}^{\infty} f(t) \sin \omega t dt$$

Rodion Tikhoplav - Vibration measurements at the A0 laser room

Harmonic analysis - Postprocessing



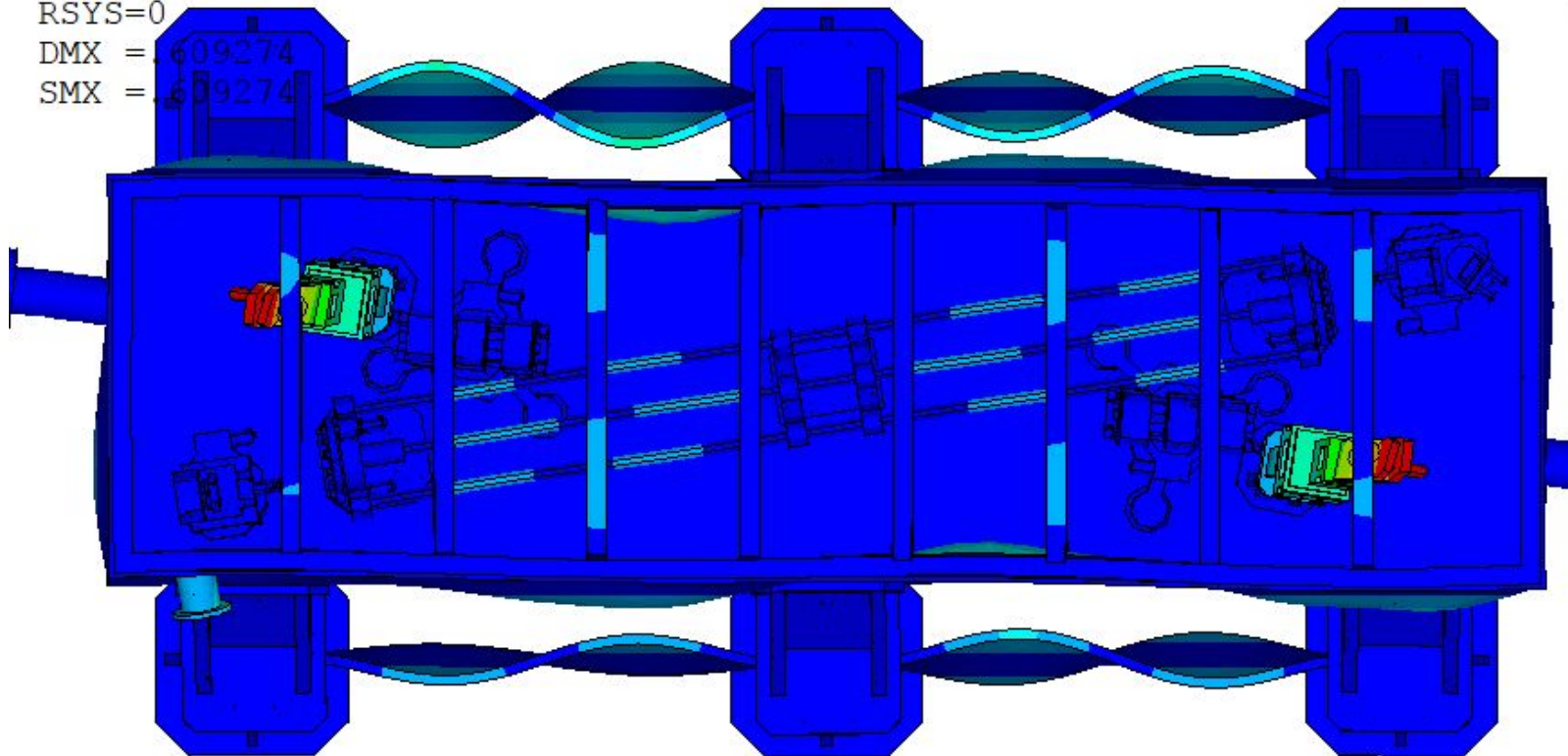
FREQ=24.0892

USUM (AVG)

RSYS=0

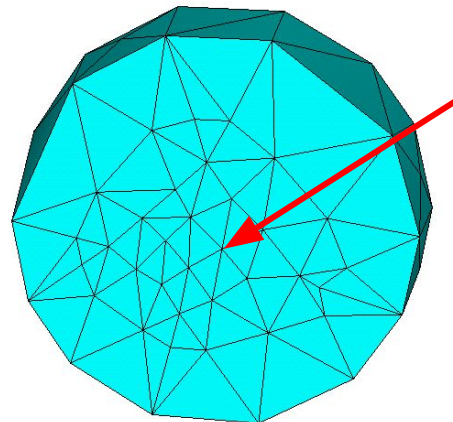
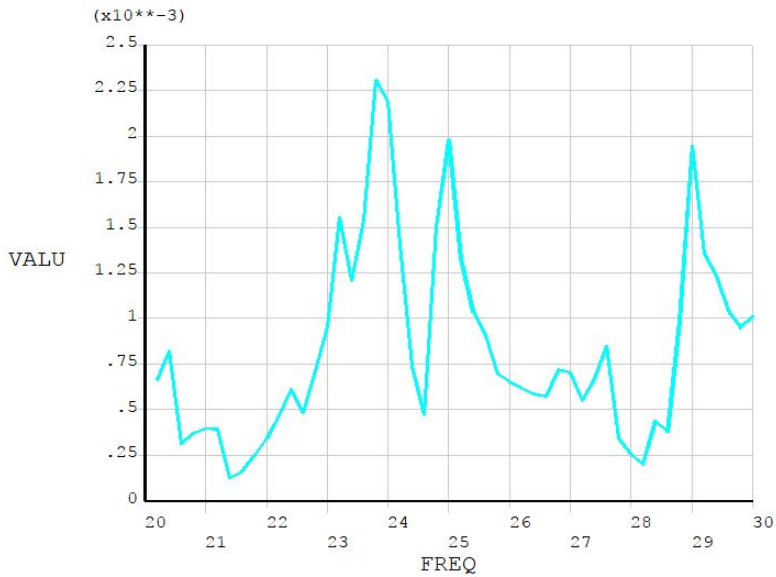
DMX = 609274

SMX = 609274

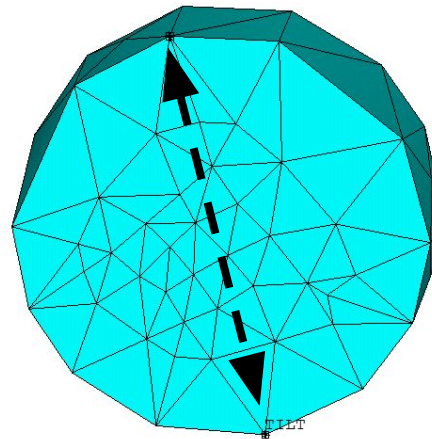
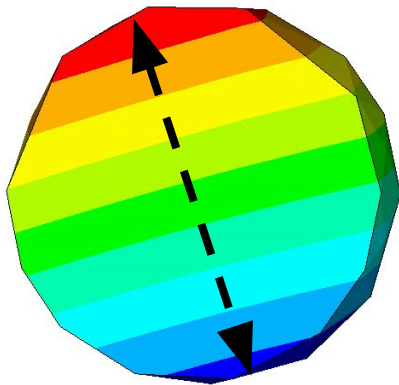


Dangerous mode to be examined - concave mirror supports

Harmonic analysis - Postprocessing



Tracking displacement of a single node over the whole frequency region in order to find the peak response



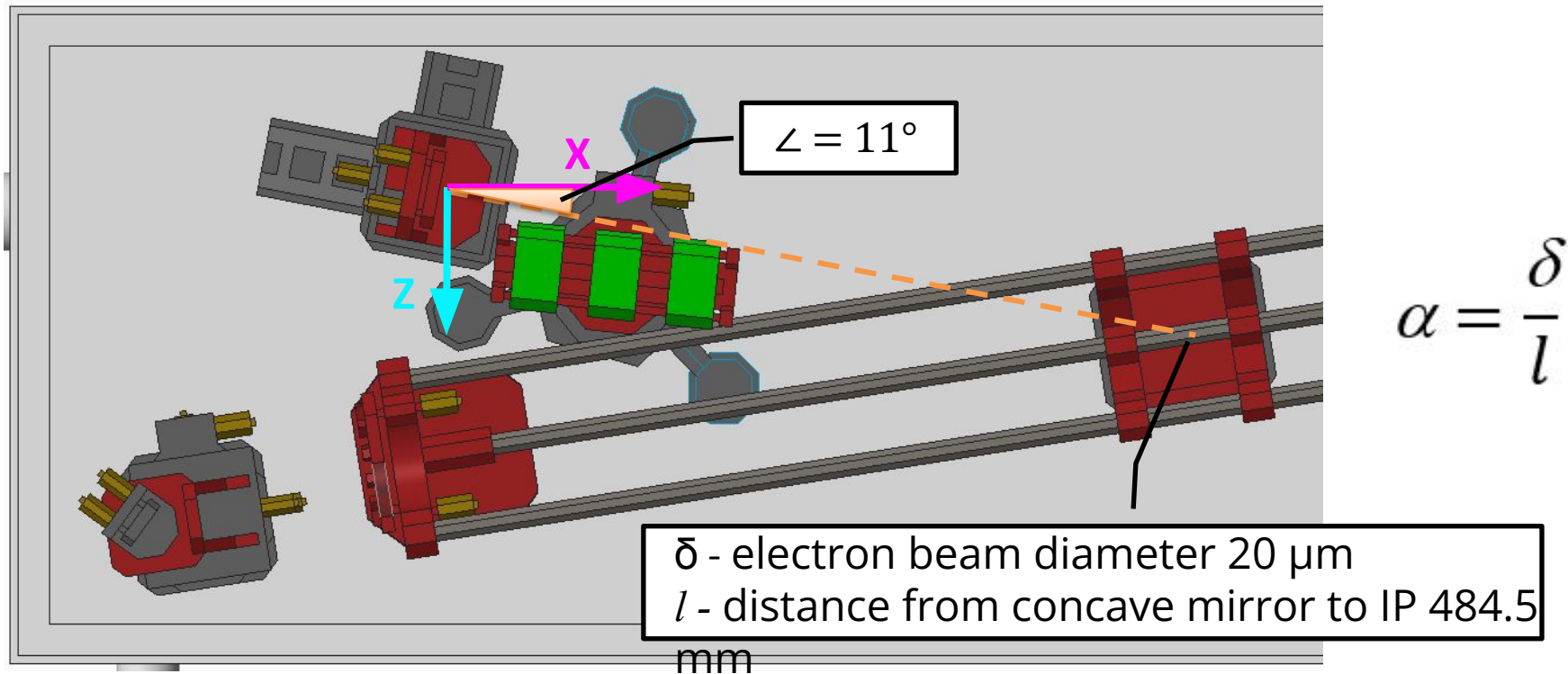
On a chosen frequency map the displacement on the path on the surface of the mirror. Linear approximation will give the tilt angle of the mirror.

Harmonic analysis - Critical displacement

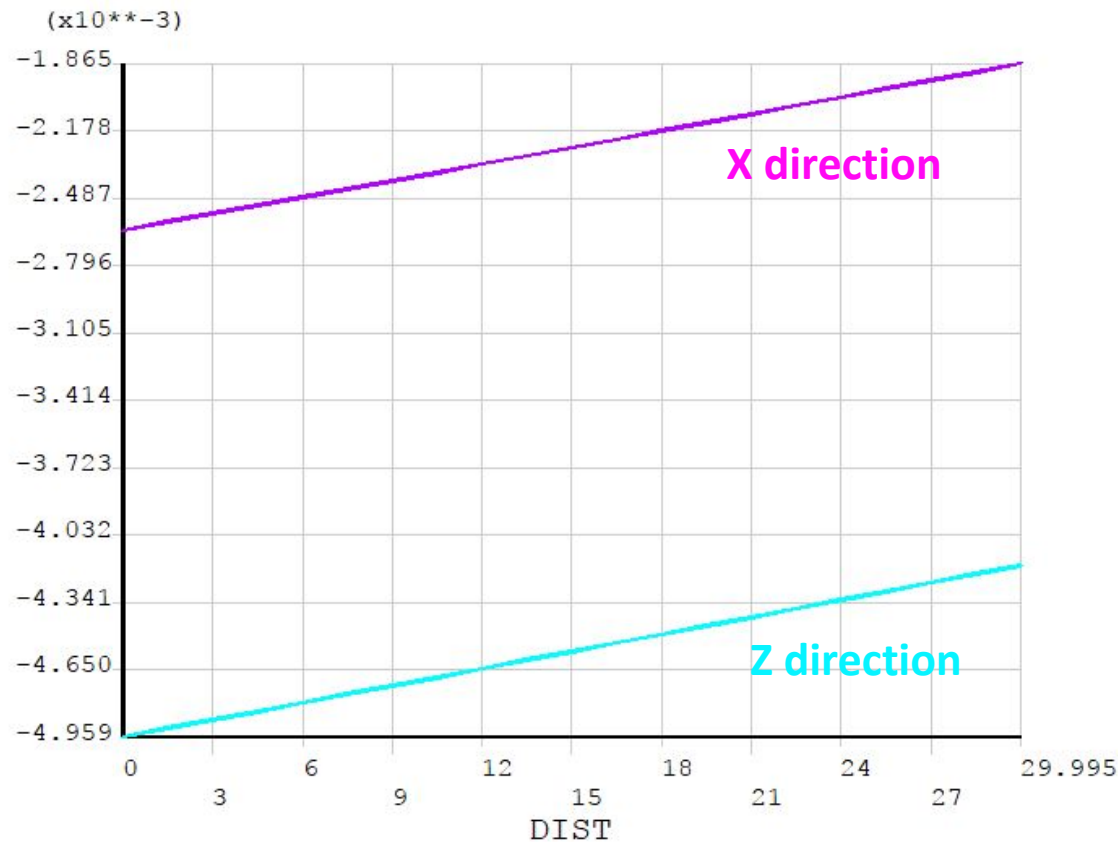


Design success criteria:

- Mirror displacement should not exceed wavelength of $1.054 \mu\text{m}$
- Concave mirror tilt angle should not exceed $\alpha = 4.13 \times 10^{-5} \text{ rad}$

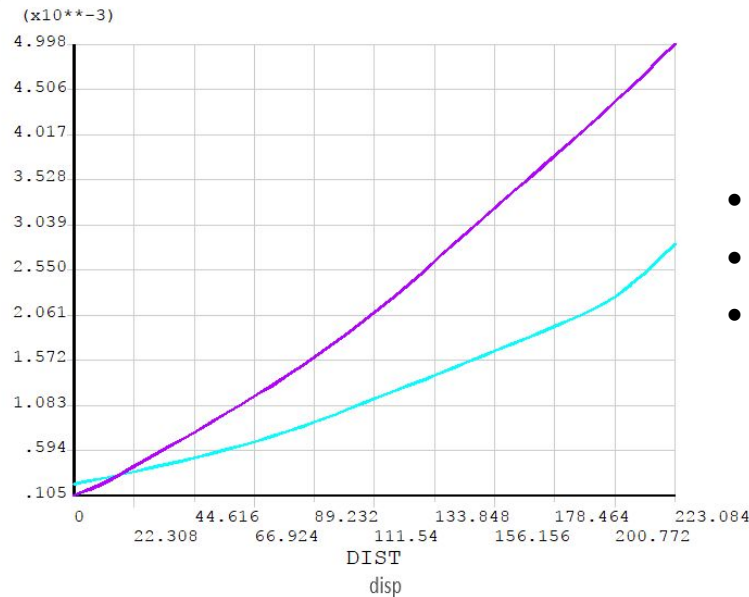
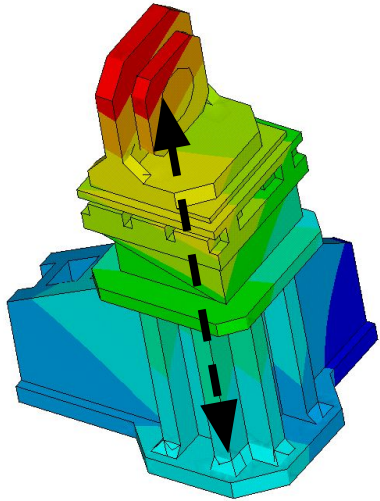


Harmonic analysis - Postprocessing

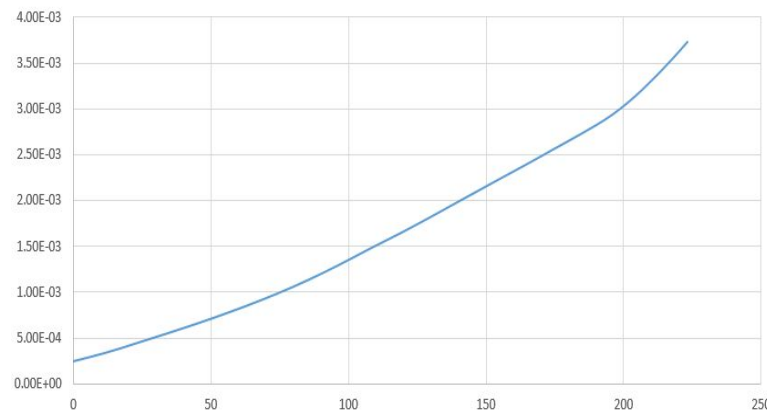
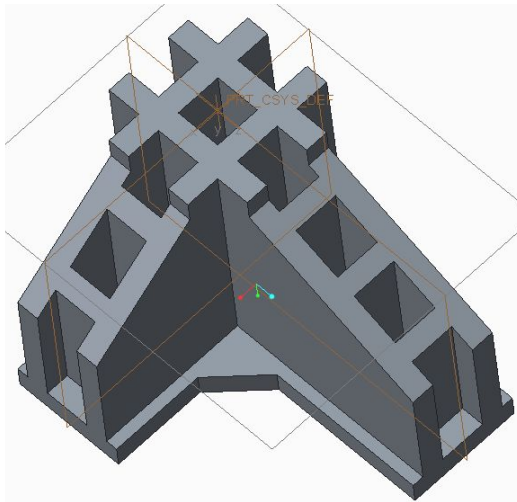


	A	B	C	D	E	F
1	S	X	Z	angle	disp	
2	0	-2.63E-03	-4.96E-03	0.191986218	3.53E-03	
3	1.4997	-2.59E-03	-4.92E-03		3.49E-03	
4	2.9995	-2.56E-03	-4.88E-03		3.44E-03	
5	4.4992	-2.52E-03	-4.84E-03		3.40E-03	
6	5.999	-2.48E-03	-4.80E-03		3.35E-03	
7	7.4987	-2.44E-03	-4.76E-03		3.31E-03	
8	8.9984	-2.41E-03	-4.73E-03		3.26E-03	
9	10.498	-2.37E-03	-4.69E-03		3.22E-03	
10	11.998	-2.33E-03	-4.65E-03		3.17E-03	
11	13.498	-2.29E-03	-4.61E-03		3.13E-03	
12	14.997	-2.25E-03	-4.57E-03		3.08E-03	
13	16.497	-2.21E-03	-4.53E-03		3.04E-03	
14	17.997	-2.18E-03	-4.49E-03		2.99E-03	
15	19.497	-2.14E-03	-4.45E-03		2.95E-03	
16	20.996	-2.10E-03	-4.41E-03		2.90E-03	
17	22.496	-2.06E-03	-4.37E-03		2.86E-03	
18	23.996	-2.02E-03	-4.33E-03		2.81E-03	
19	25.496	-1.98E-03	-4.29E-03		2.77E-03	
20	26.995	-1.94E-03	-4.25E-03		2.72E-03	
21	28.495	-1.90E-03	-4.21E-03		2.67E-03	
22	29.995	-1.87E-03	-4.17E-03		2.63E-03	
23						
24	3.00E+01	max-min			9.03E-04	max-min
25						
26	l	4.845E-01	4.128E-05 rad		3.01029E-05 rad	
27	delta	2.000E-05				
28			1.054E+00 μm		3.530606178 μm	

Harmonic analysis - Solutions



- Geometry modifications
- Extra supports
- Make shorter mounts



Height support modification has mitigated maximum response in the mirror from 7 μm to 3 μm

Conclusion



- ICS is an exceptional method of generating γ radiation of high brilliance, its development is important for National security and a number of other applications.
- Designing of ICS interaction region is a complicated process that comes in several interconnected stages.
- Present design is a trade-off between technical requirements of finesse, size, mechanical stability and overall complexity. It has its limitations.

Thank you for your attention

